

EH Characterisation

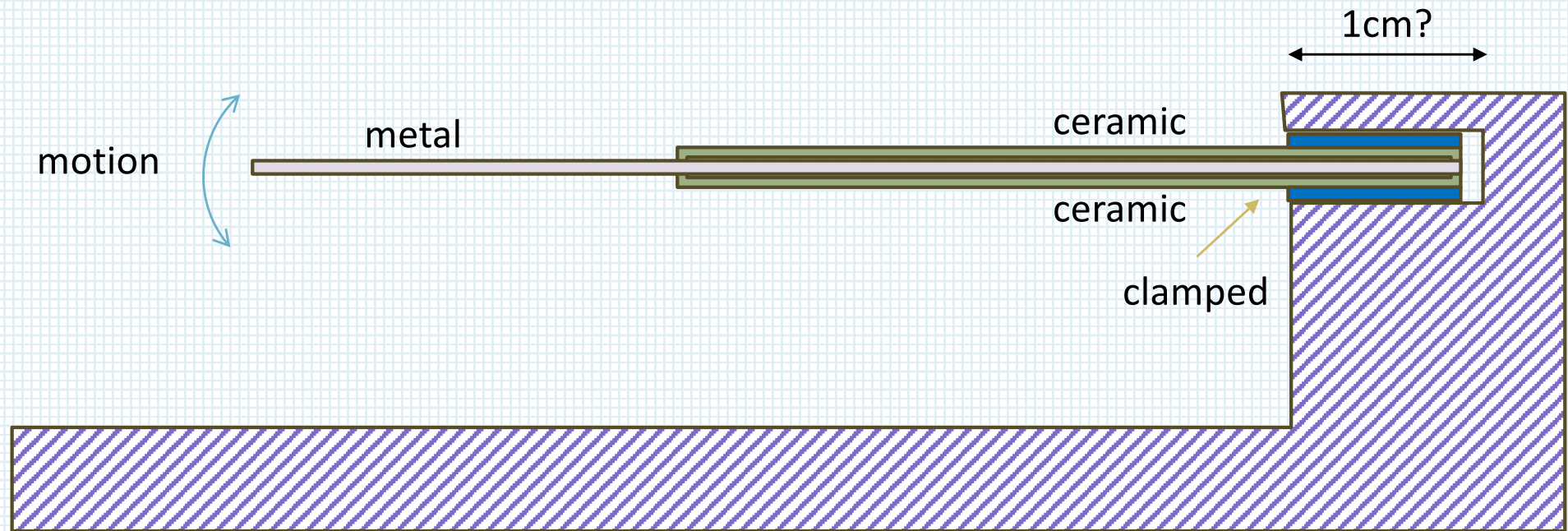
Experimental plans – LSBU project file

Potential Energy Group: PO 001

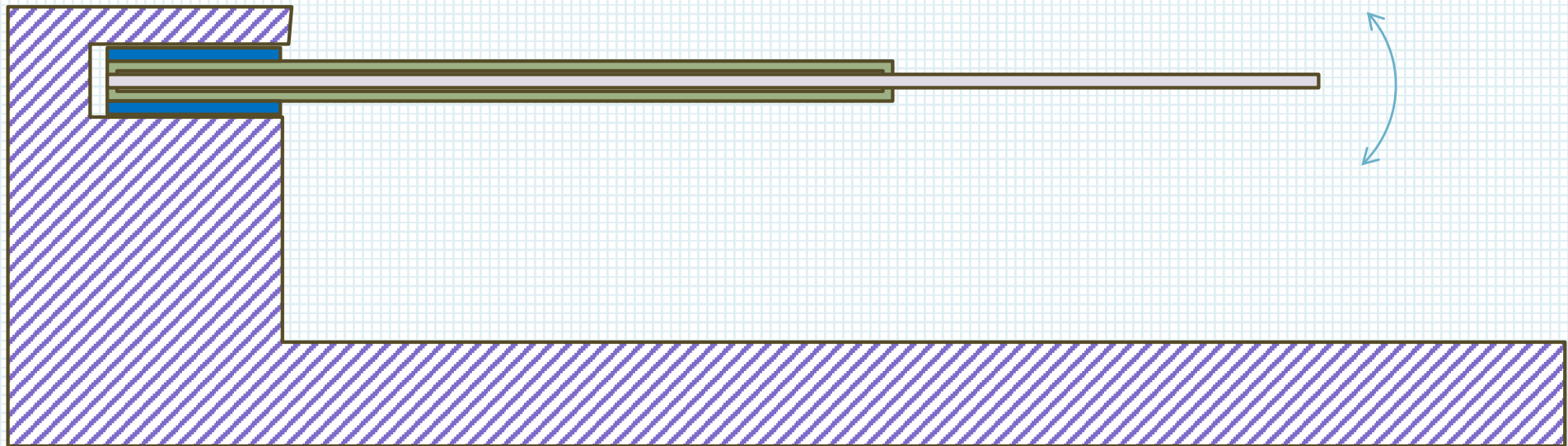
Our ref: ESQ1980

Nov 24-Dec 24

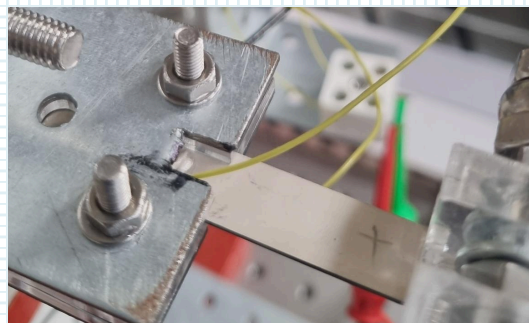
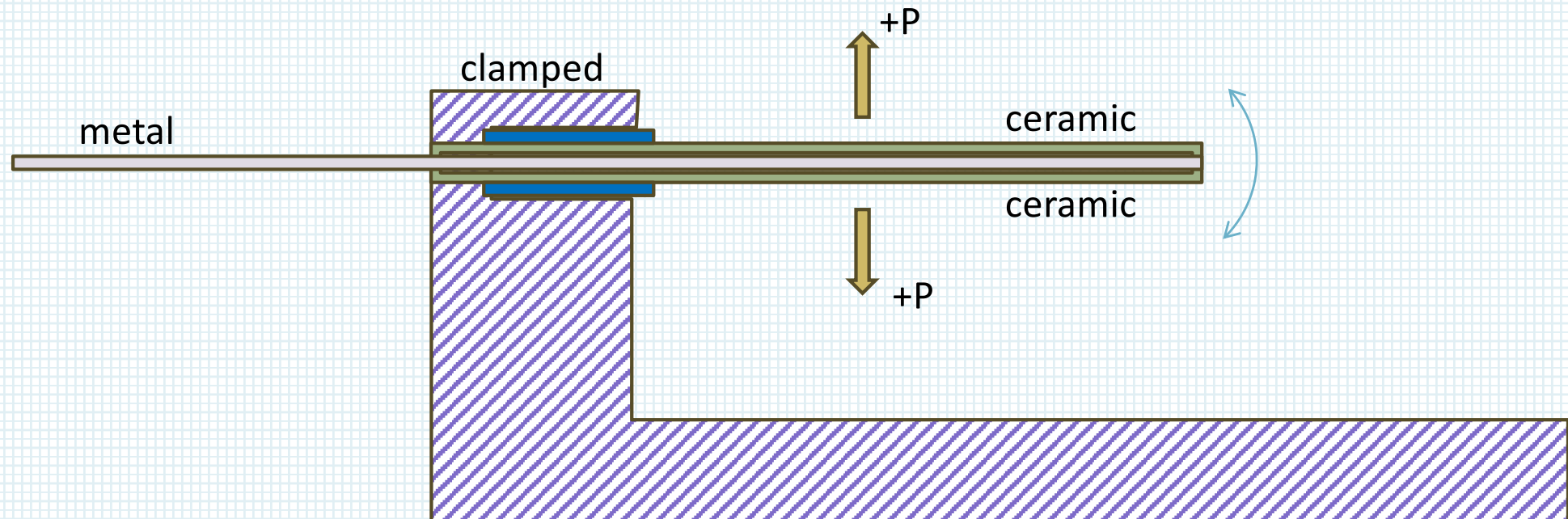
Schematic 1



Schematic 2

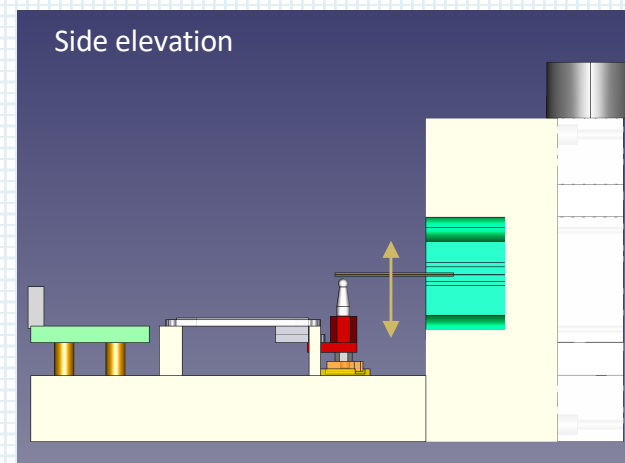
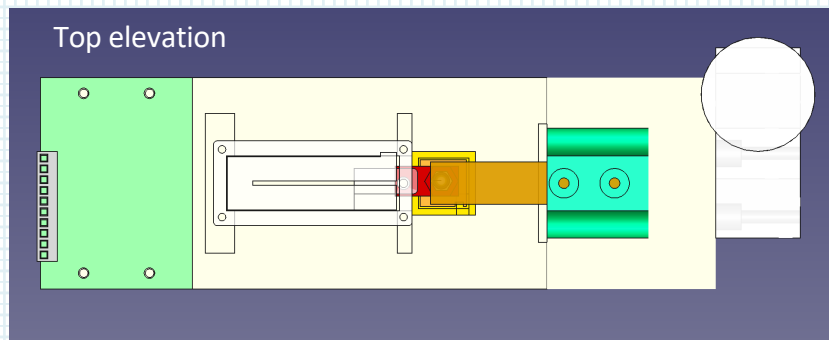
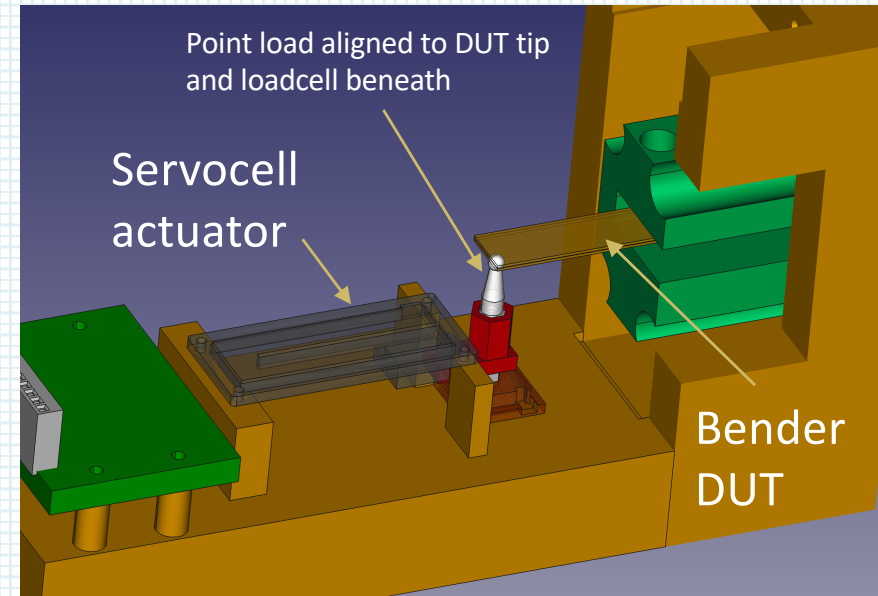
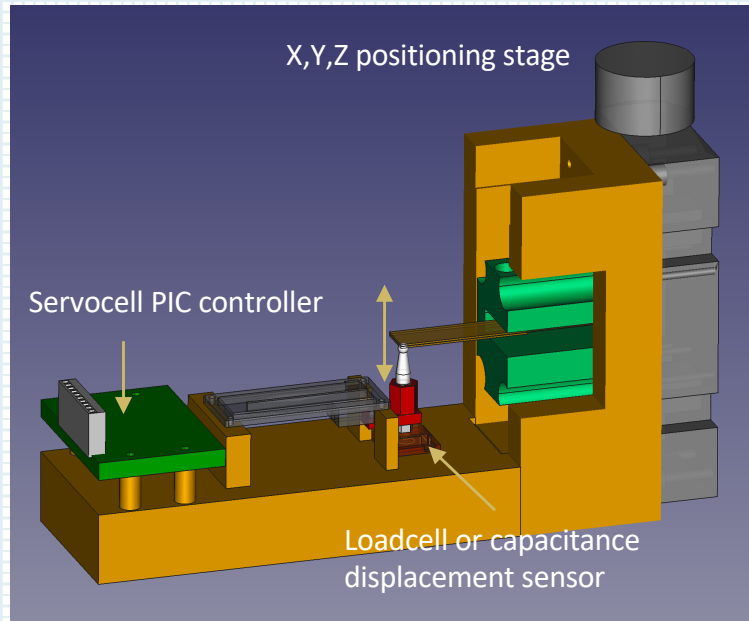


Schematic 3

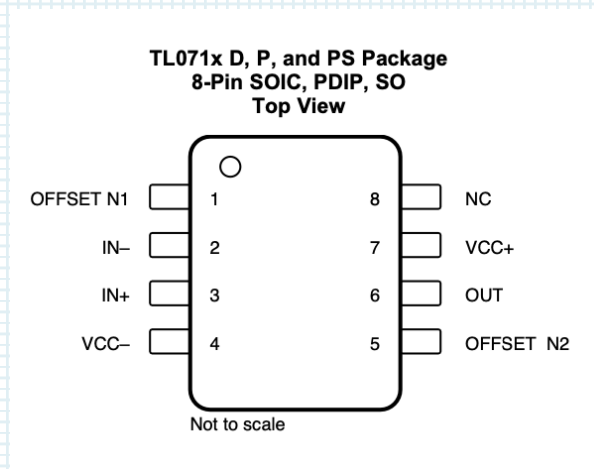
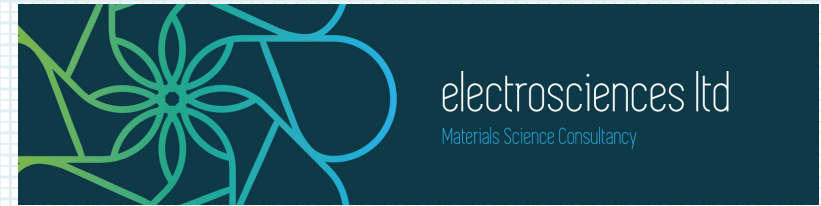


- Two ceramic layers are oppositely poled so positive polarisation pointing out.
- Metal shim central connecting electrode.
- SCD does not use the metal shim. Just connecting the two yellow wires to voltmeter

System Build



Electronics buffer

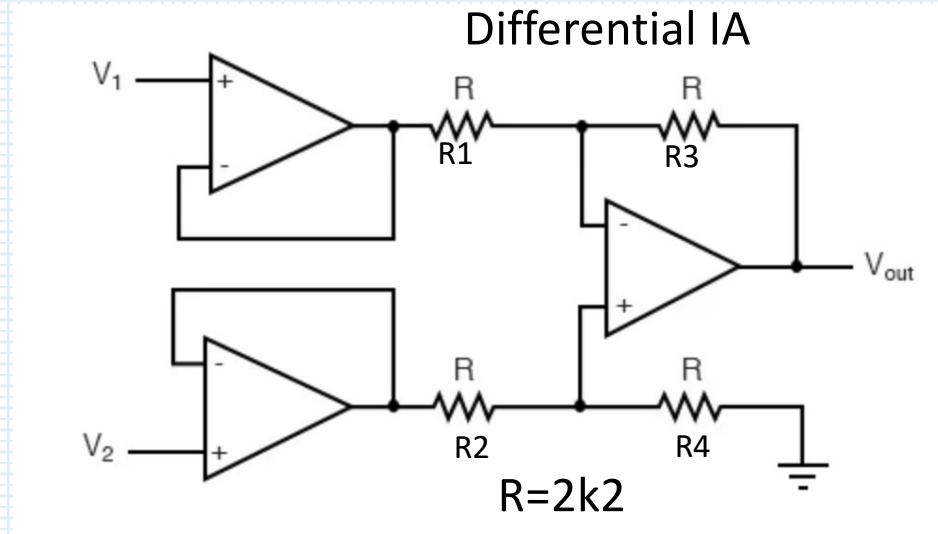


TL071 ±10V

Check: Gain: 1.6

INPUT: 200mV rms

OUTPUT: 323mV rms



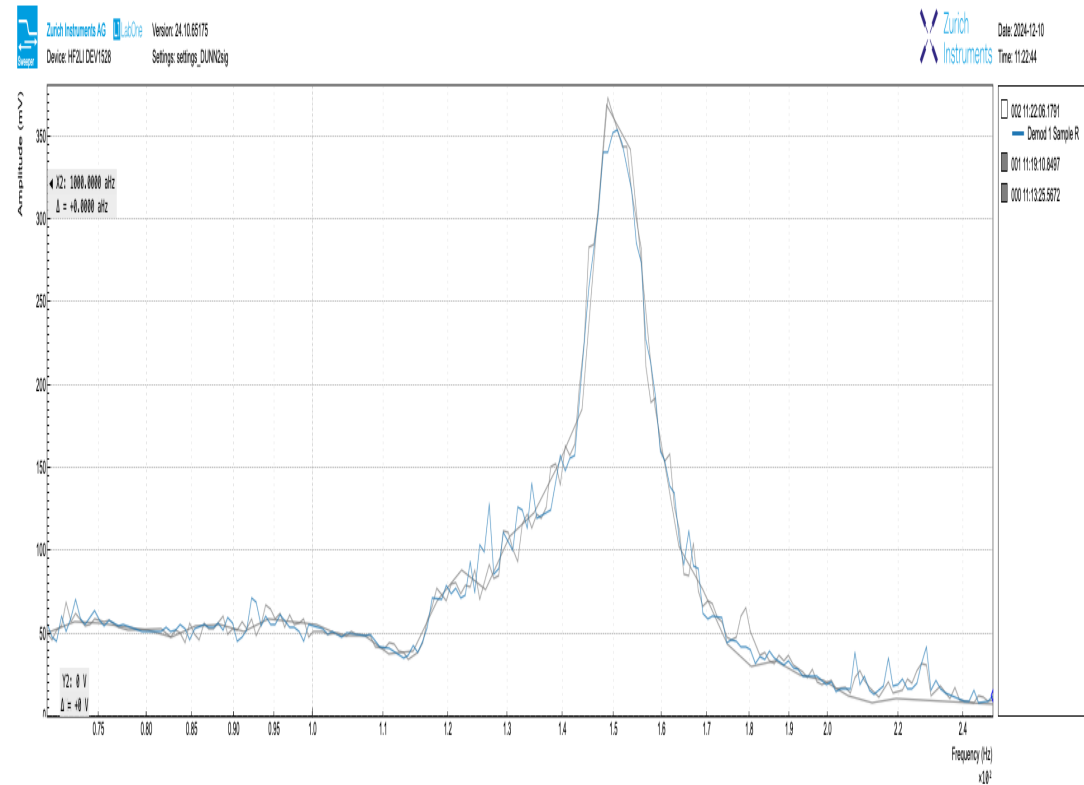
Gain: unity:

$$V_{out} = (R_3/R_1) (V_2 - V_1)$$

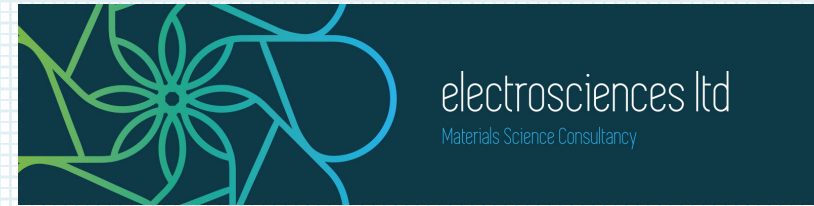
- R_L as load for bender element: can exceed the input resistance of modern ADC / input amplifiers / oscilloscopes. This changes the effective load on the bender.
- Solution: pass output of R_L loaded bender through a buffer (voltage follower) Differential Instrumentation amplifier: output impedance many hundreds $M\Omega$

FRA Lock-In Initial analysis

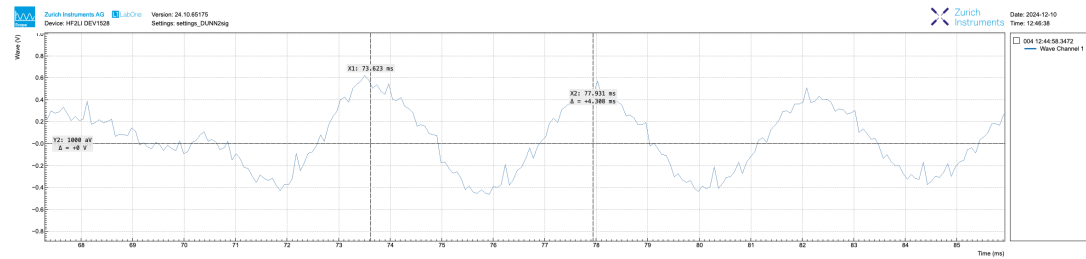
- 4M Ω load on the piezo bender
- Use of 2 buffer amps for each lead from bender
- Buffer amps bipolar supply. TL071 JFET input very high input impedance $\gg 10^{12}$ Ω
- Buffer chips feed to unity gain non-inverting amplifier.
- Power supply: $\pm 10V$
- Lock-In: Zurich Inst HF2LI
- (initially used the MOKU Go but the outputs had large offset)
- OSC 2 to sig o/p 2.
- Settings: saved DUNN, settings_DUNN2sig
- Inputs: DC, HighZ, single ended.
- Outputs: 2.5V, offset 2.5V sine ON.
- Resonance around 150Hz. This may be servocell resonance or sample.



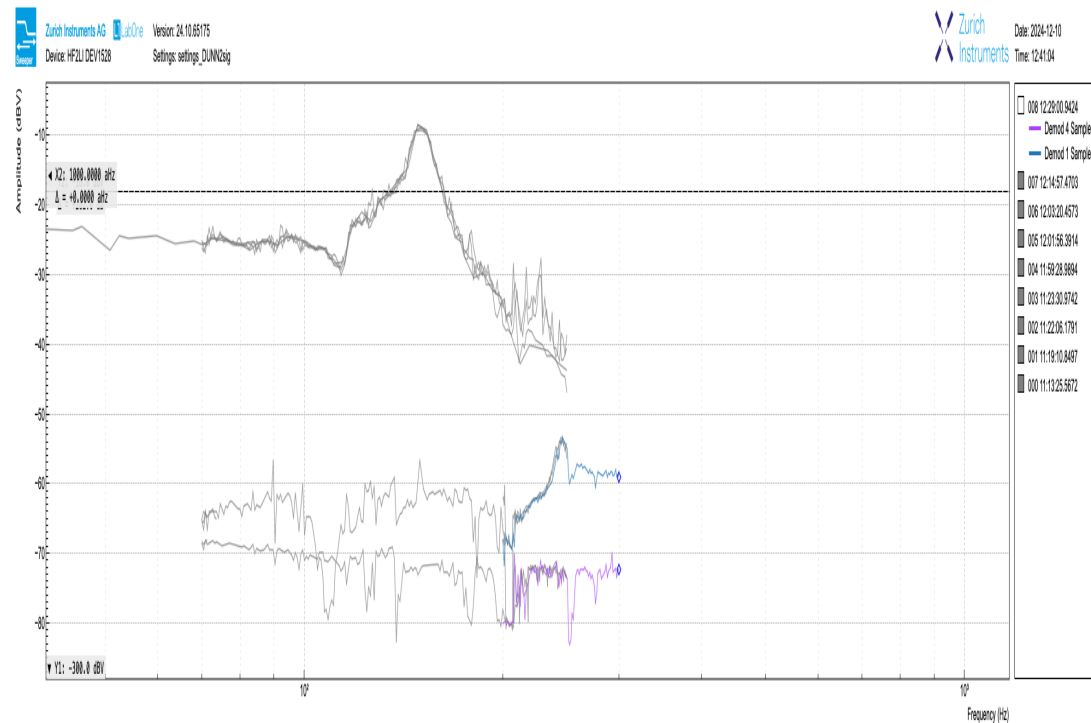
Loaded bender with loadcell: resonances



Free resonance 233Hz: Ping on bender



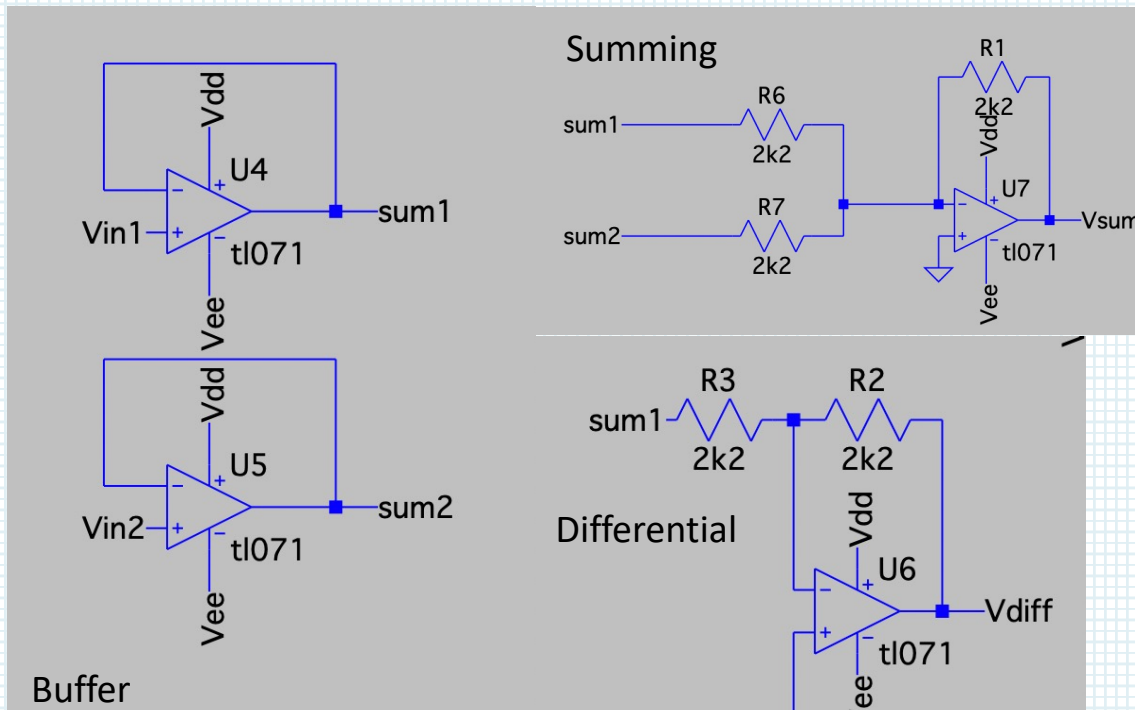
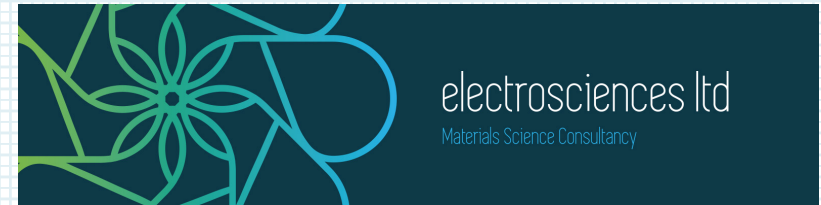
Freq scan with no loadcell: fr~150Hz



Loaded with loadcell: resonance peak around 246Hz

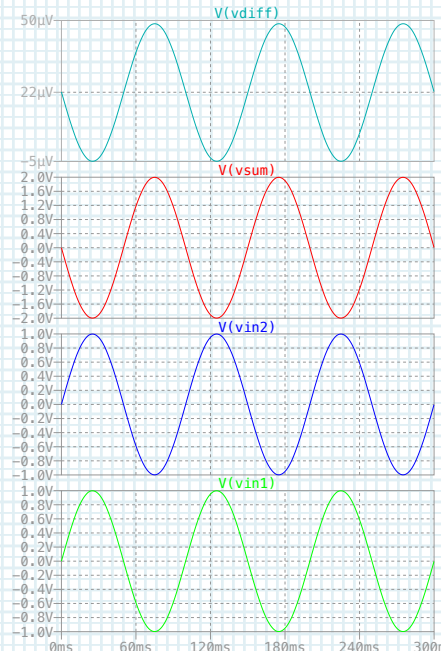
Buffer:

Difference and Summing instrumentation amplifier

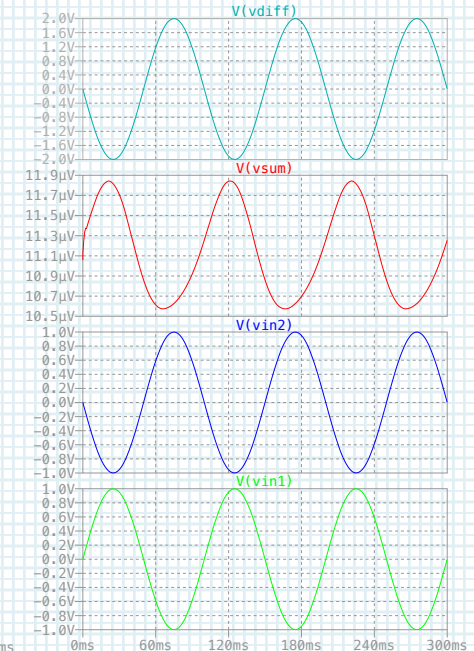


LTSpice: TL071 opamp: TL071.301
/Users/markyscain/Documents/Science_Programs/LTs
pice/Energy_Harvesting/EH_buffer1.asc

In Phase



180° out of Phase



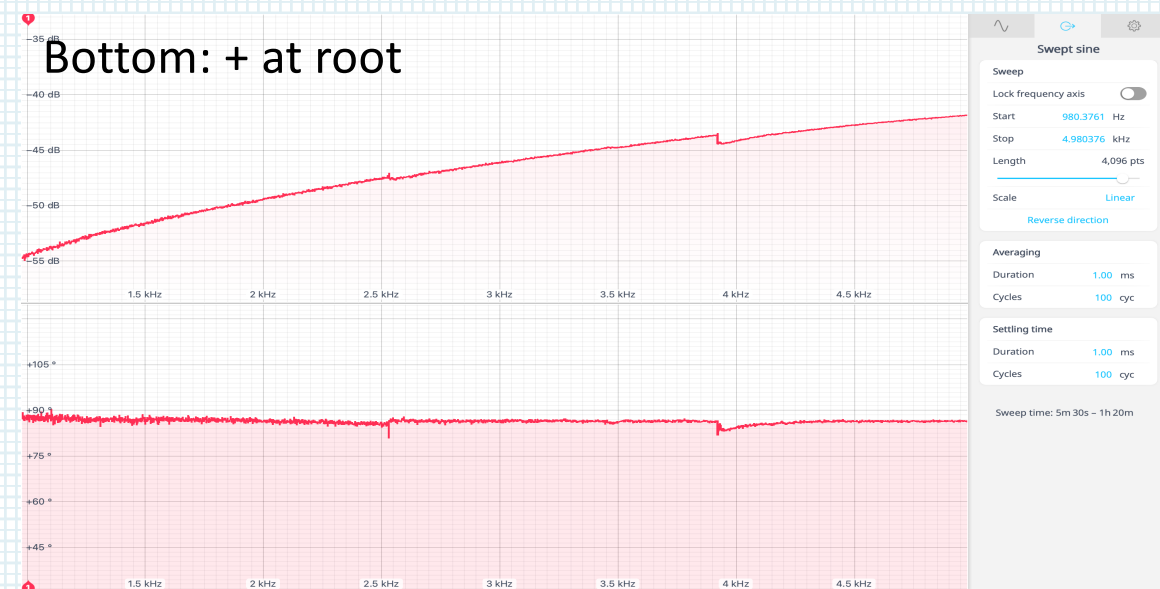
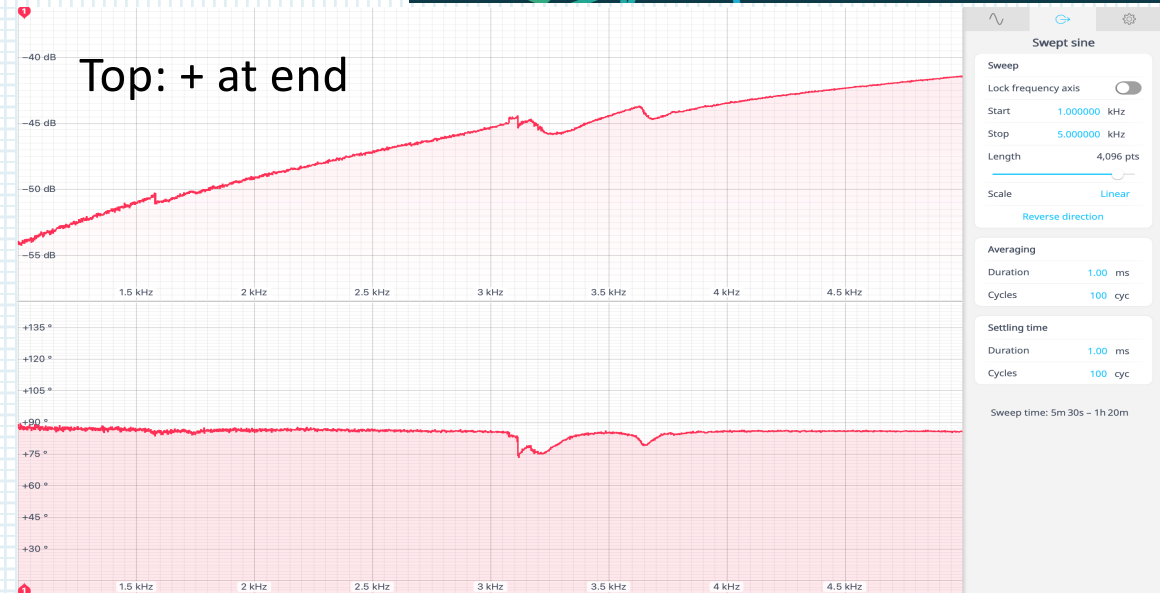
Outcome:

- In Phase signals, we need SUM
- Out of-phase signals, we need DIFF

LTSpice circuit analysis and response

Electrical resonance of bender

- Individual addressed resonance of each ceramic plate
- Similar resonance behaviour (MOKU-GO –FRA). i/p: 50 ohm terminated.
- o/p: 200 ohm
- Positive (biased sine wave) to ceramic plate.
- Resonance of both plates different. Implies coupling or thickness of each plate is different – or material is different.
- Repeated in later slide.
- /Users/markyscain/Documents/Electrosiences/DUNN EH 2024/Experimental Research/FRA/



Electrical resonance

MOKU GO: FRA

Connections through top and bottom ceramic layers, through 2 yellow wires.

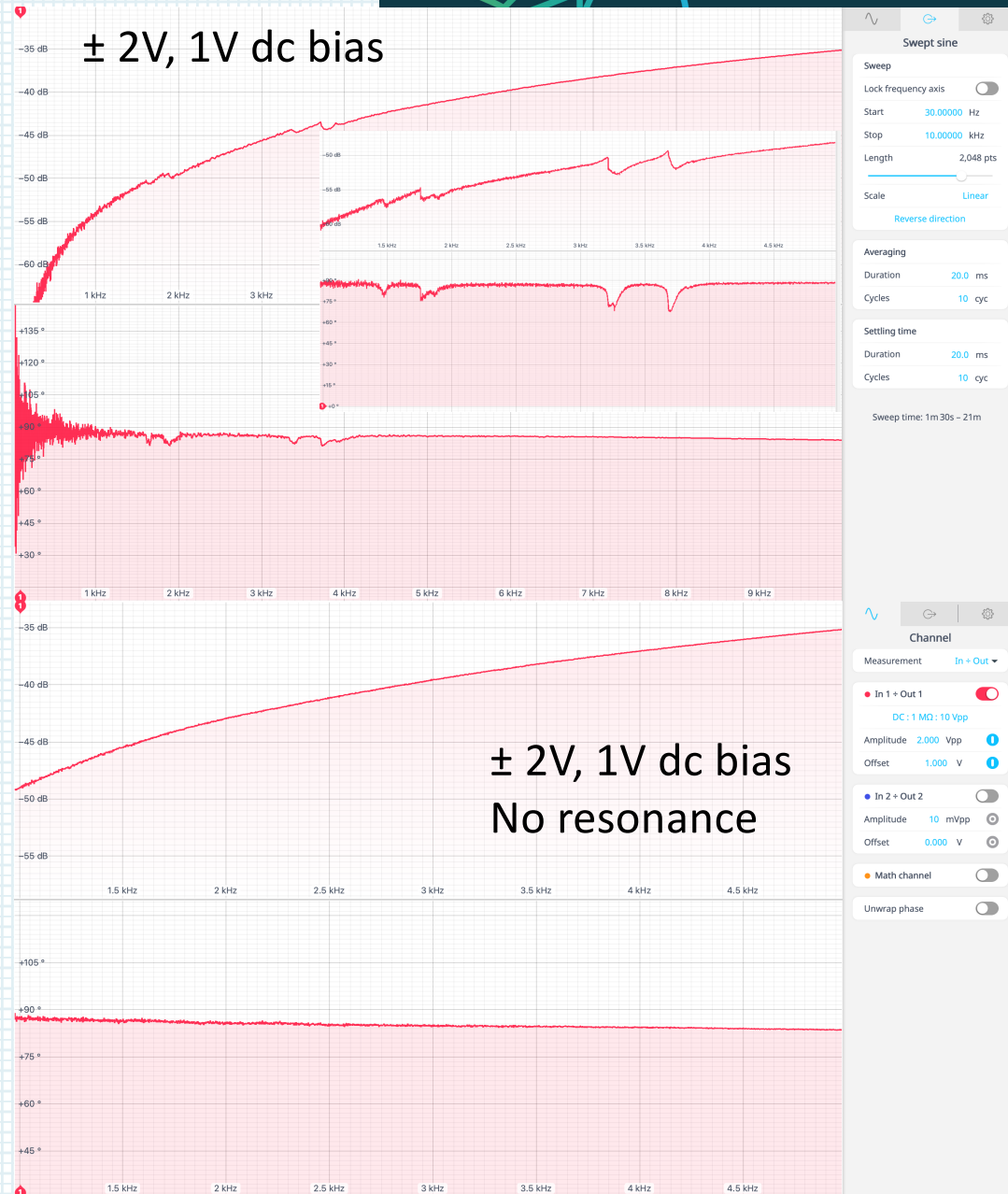
Excitation:

$\pm 2V$, 1V dc bias

Two examples.

Both ceramic sides driven together:
current taken from metal shim

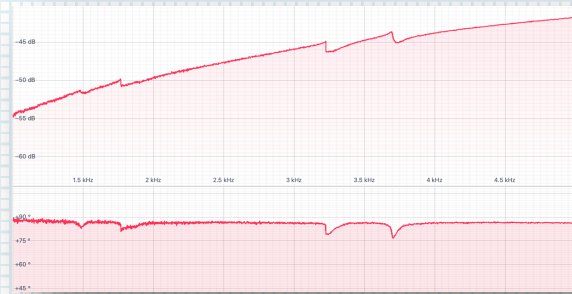
$\pm 2V$, 1V dc bias



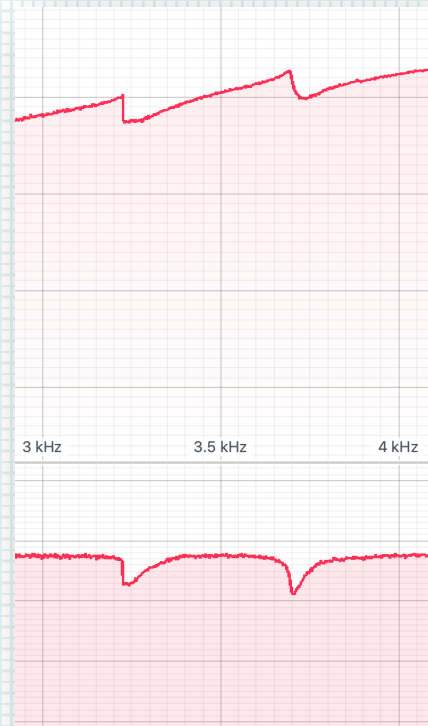
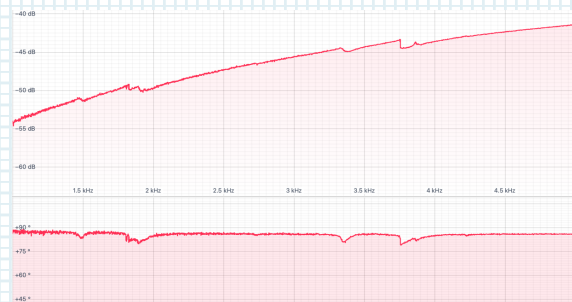
Electrical resonance – separate ceramic sheets

2Vpp, sine, 1V DC. Positive always to ceramic.
 Metal shim return path to FRA Input 1.
 Similar, but not identical resonances.

Lower



Upper



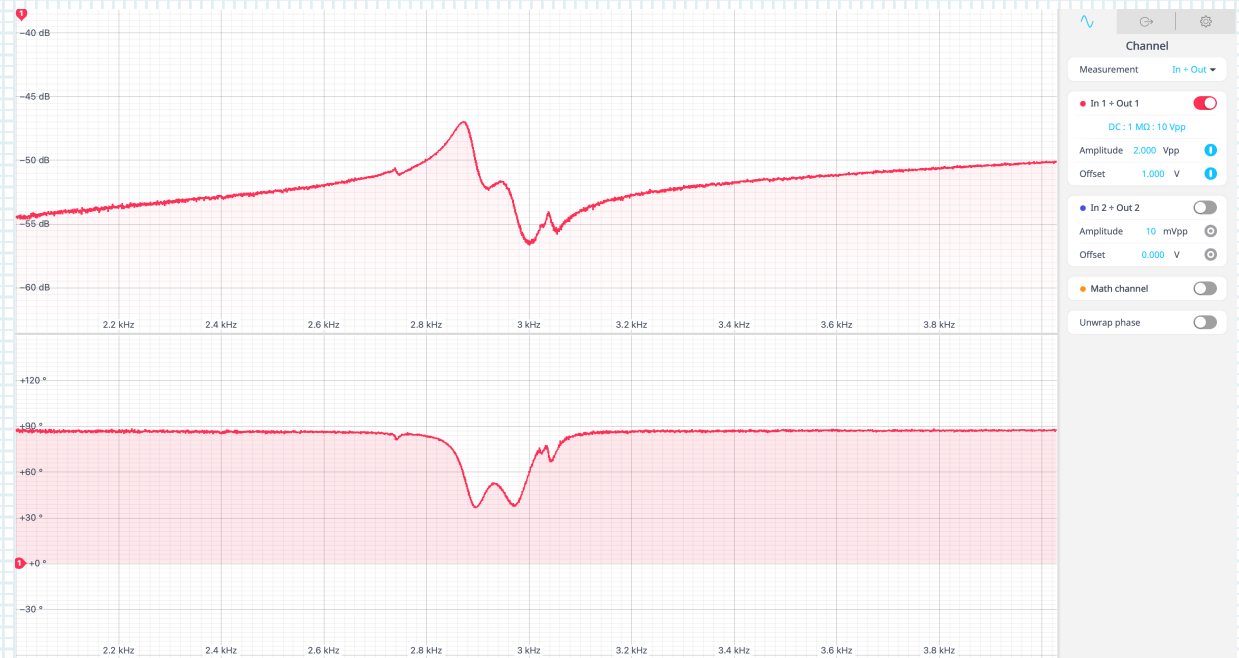
Lower



Upper

Electrical resonance FRA

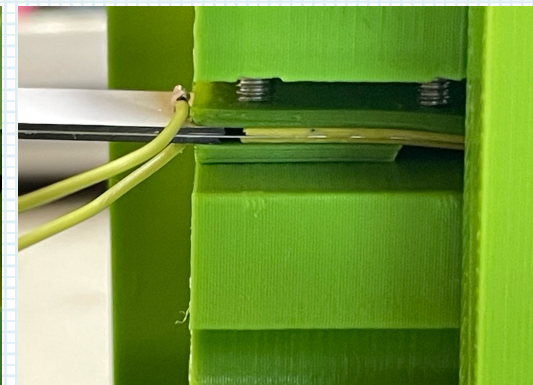
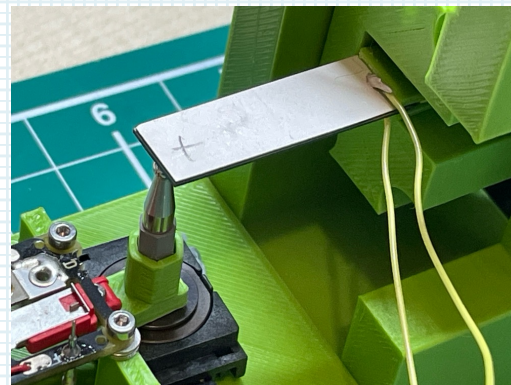
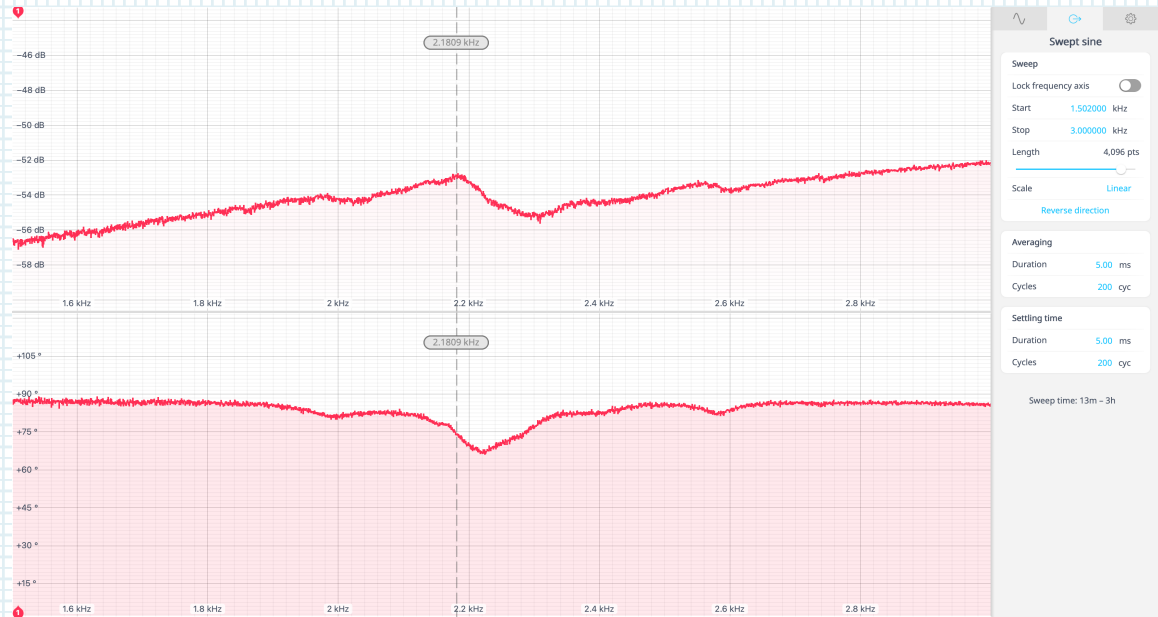
- HPZ actuator free resonance. held by inner metal shim. Ceramic in air, not touching anything. yellow wire in, yellow wire out. No RI load. Resonance different to when actuator resting on surface.
- Clean resonances – 2 off based on slight difference in top and bottom ceramic.
- 2.8702kHz
- 2.951kHz
- Tiny at 3.0399kHz and 2.7382kHz



'/Users/markyscain/Documents/Electrosiences/DUNN EH 2024/Experimental Research/FRA/MokuFrequencyResponseAnalyzerData_20241216_195612_Screenshot.png'

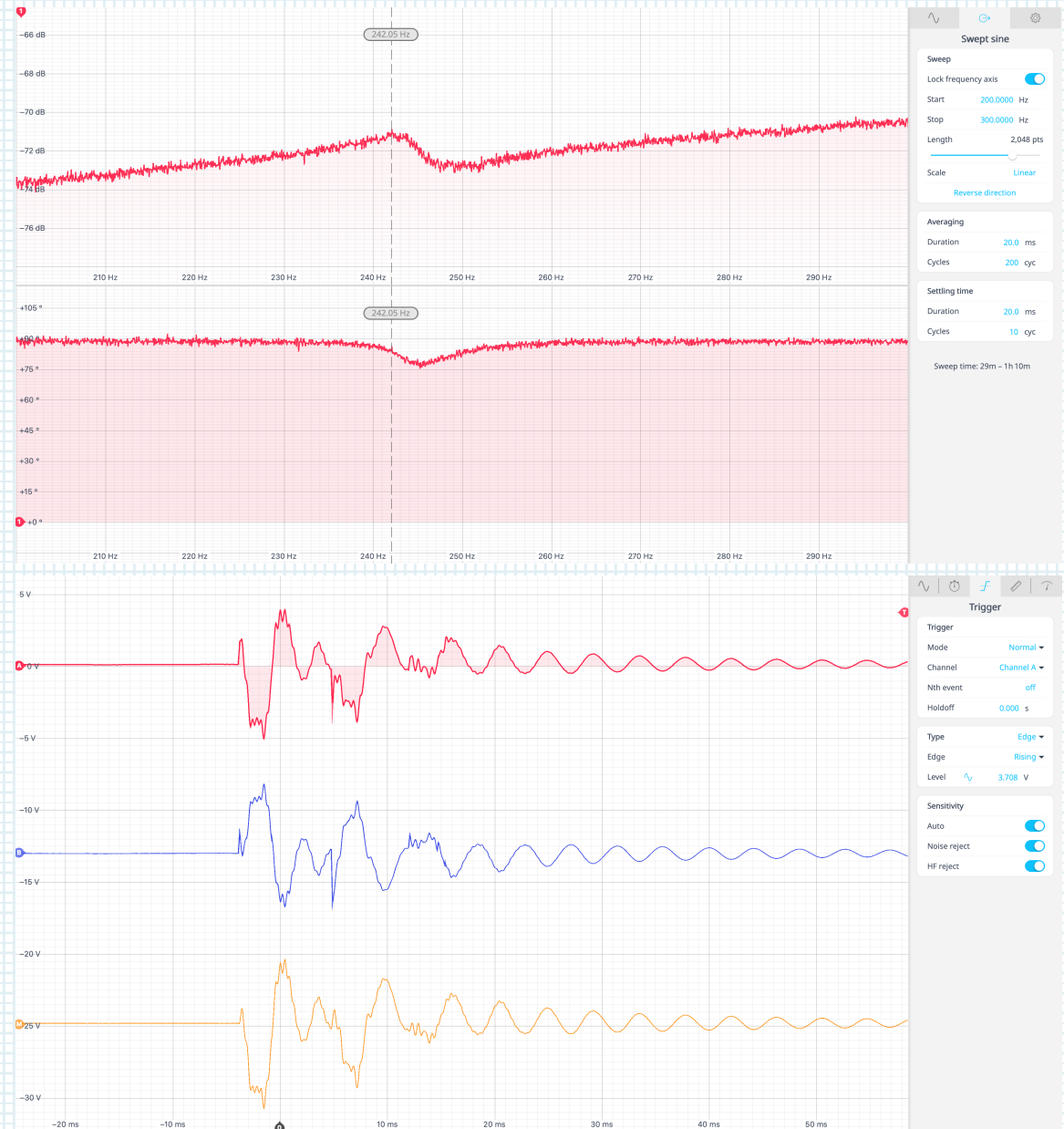
Clamped resonance

- Clamp bender using ABS packing pieces.
- Resonance for thickness mode.
- $L_0=38\text{mm}$
- $F_r \sim 2.2\text{kHz}$
- Increase clamp Allen key bolt slightly: no change in fr.



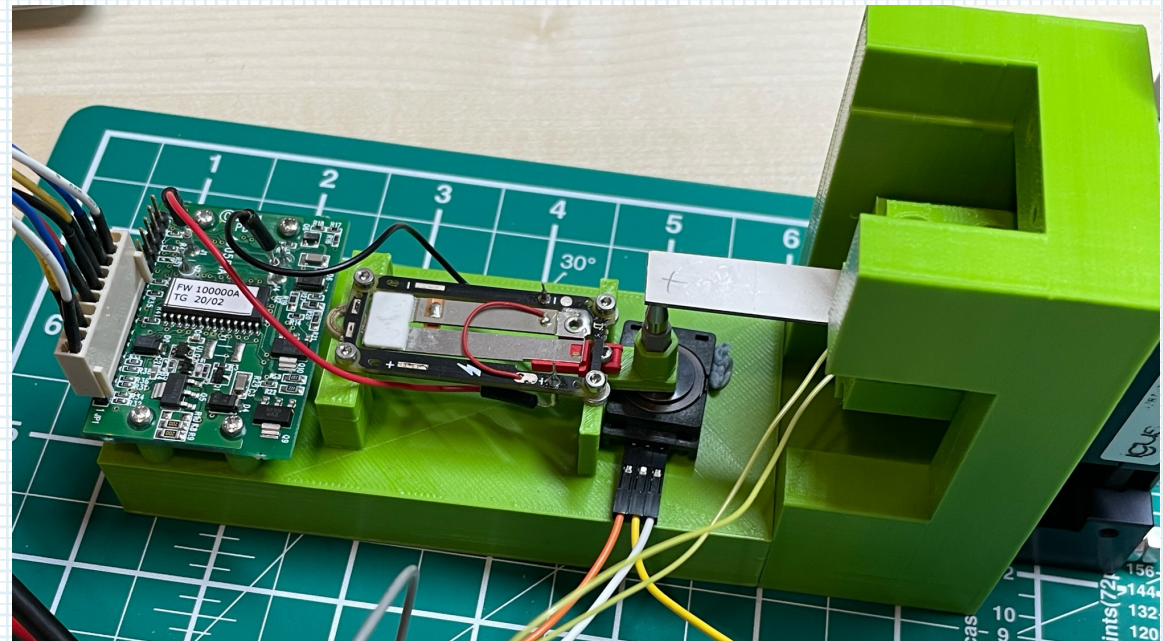
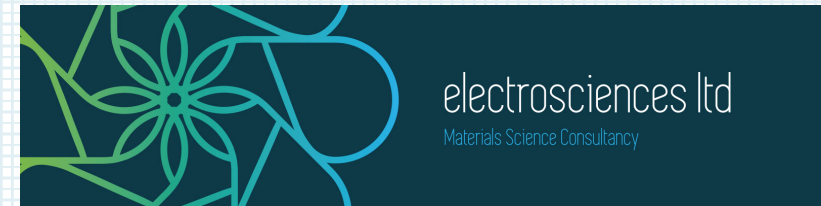
Bender resonance

- Bender resonance:
 - driven using FRA at $\pm 3V$ dc offset: $f_r \sim 242\text{Hz}$.
 - Mechanical ping: oscilloscope capture: $f_r \sim 241\text{Hz}$
- Run at f_r mechanical. 241 Hz



System : small signal excitation, R_L loading

- Zurich Instruments: HF2LI lock-in(LI) amplifier.
 - Excitation: $\pm 5V$ pkpk, 2.5V dc biased.
 - Off resonant response.
 - Bender and R_L (R_{load}) element routed through differential IA (buffer).
 - Loadcell as second input to LI
 - Lock-in data is recorded and logged as a function of R_L and operating frequency.
-
- Plot: power generated against R_L and against force.
 - Acceleration calculated from bender displacement via:
 - force, frequency, span length and PZT modulus.
 - Capacitance displacement sensor, micrometers.



Force:

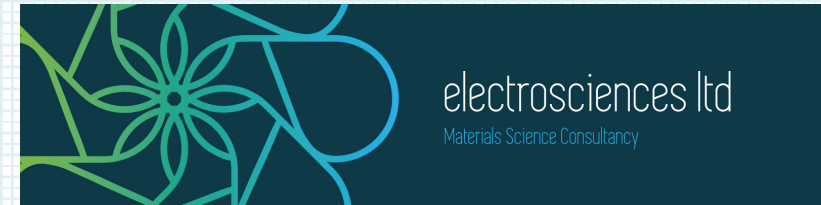
Calibration mass: 100g
 TE Connectivity load cell FS2050-0000-1500-G (RS 893-7225).
 1500gF full range at 3V span (5V supply)
 $1500gF = 14.7N$
 $F = ((V_0 - V_z) / 3.0) \times 1500 \times 9.81 / 1000 [N]$
 $V_z = 0.988V$

Calibration: 100gF (0.98N): $V_0 = 1.183V$:
 $F = 0.96N$

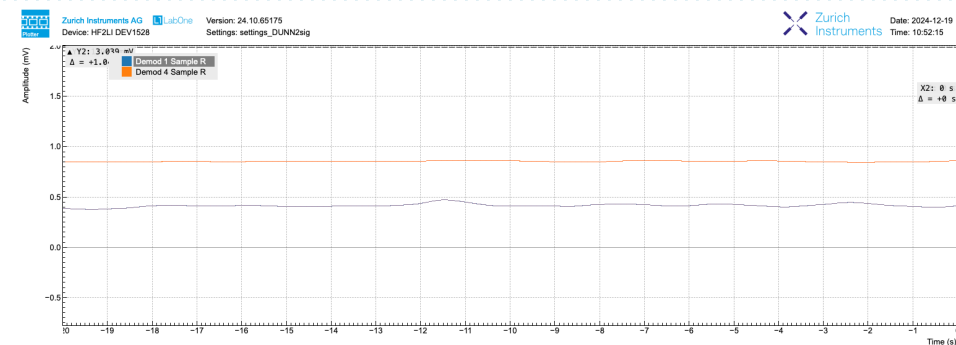
Bender Voltage output: V_B

Lock-in voltage rms: V_{LI}
 $R_L = \text{ohms}$
 G_1 : Transfer through DIFF-IA: $\times 1.6$
 G_2 : Gain: LI set = $+10$
 $V_B = V_{LI} / G_1 G_2$

Calibration – offsets: capacitive pickup/induction from cantilever

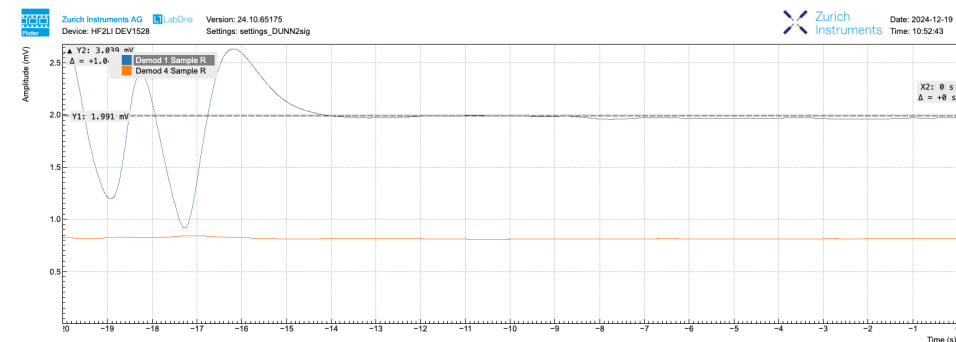


- Bender element in vicinity of driving electronics (EMI)
- Bender not touching load probe.
- No flexure, SERVOCELL EM pickup at 35Hz: 2MOhm loading: 406 μ V.
- Remove this offset on all data.



/Users/markyscain/Downloads/meas_plotter_20241219_105215.png

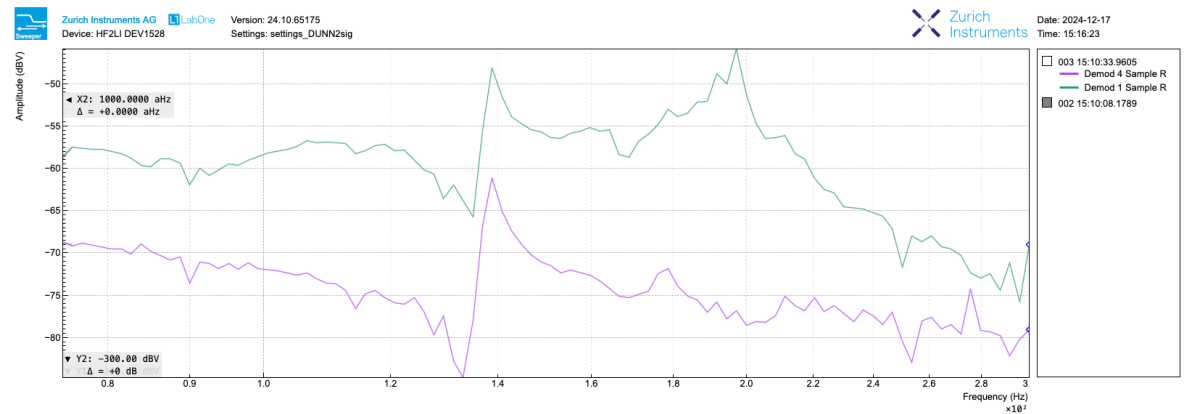
- Loading probe full contact with bender.
- Lock-in data very stable.
- Lock-In Bender V_{out} 5 times pickup.



/Users/markyscain/Downloads/meas_plotter_20241219_105243.png

Small signal response

- HF2LI output: sweeper response. Freq. scan.
 - Green: Voltage output over 1MΩ load
 - Purple: Loadcell.
- Servocell input is voltage to the actuator. Open loop control. Nothing is controlled. Hence load is changing with bender response and with frequency.
 - We can use closed loop control to control force. Or to control displacement.
 - Or, we can normalise the output with changes in load with freq.

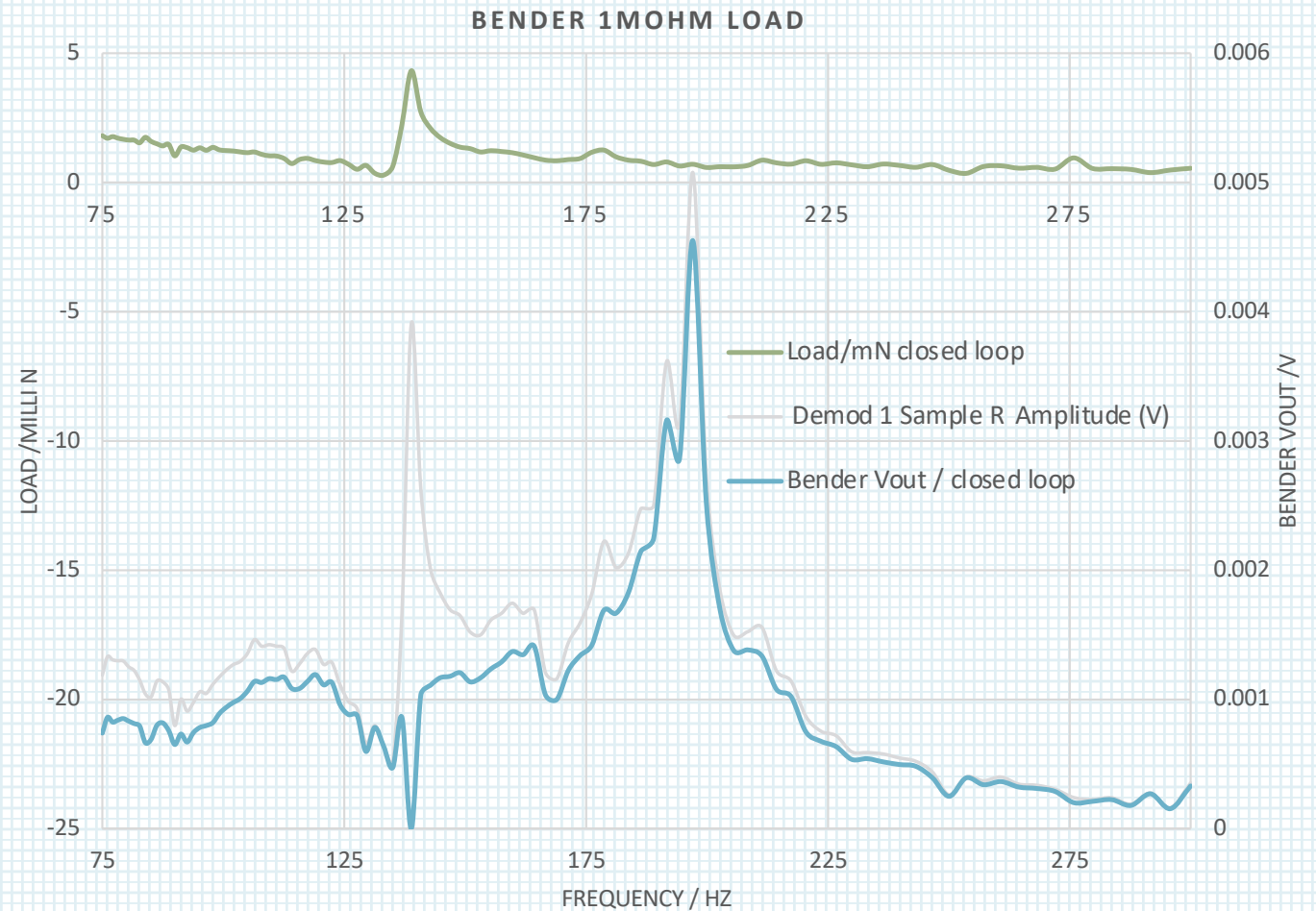


Control	Settings	History	Math
Run/Stop	Single		
Horizontal	Copy from X-Axis X-Cursors		
Start (Hz)	75.00000000		
Stop (Hz)	300.00000000		
Length (pts)	100		100%
Sweep Param	Osc 2 Frequency		
Sweep Mode	Sequential		
X Distribution	Linear Logarithmic		
Remaining	0.0s		
Plot 1	Plot 2	XY	
Demod R	1	Add Signal	
Amplitude (V)	Demod 1 Sample R		
	Demod 4 Sample R		
Aux In (V)			
Frequency (Hz)			
Phase (deg)			
Amplitude (V/√Hz)			
[Drop signal here for new group]			
[Drop signal/group here to remove]			

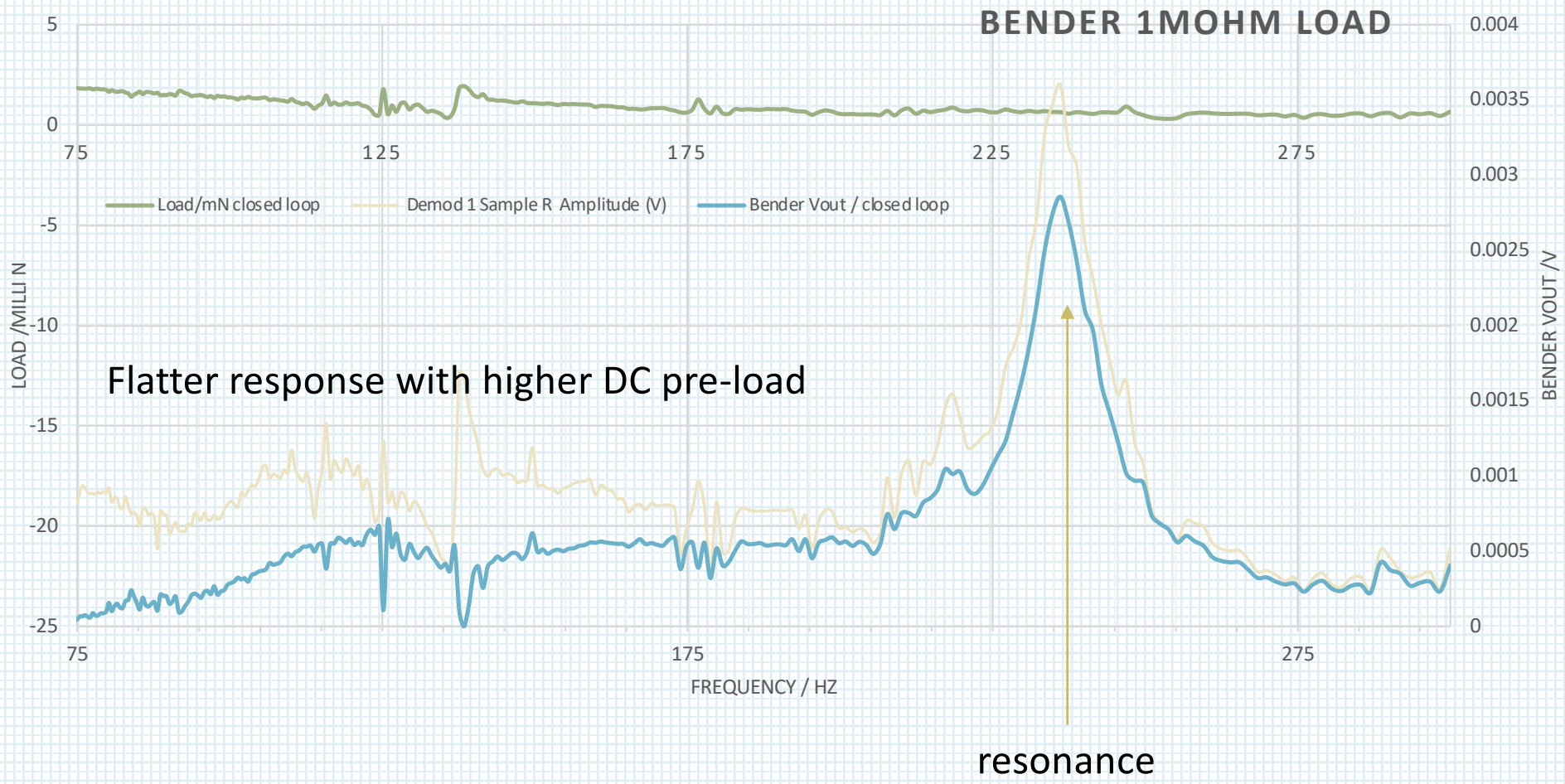
Control	Settings	History	Math
Run/Stop	Single		
Mode	Application Advanced		
BW Mode	Auto Fixed Manual		
Order	4		
Max BW (Hz)	1.250M		
BW Overlap	0		
ω Suppr (dB)	60		
Setting			
Min Time (s)	100.0m		
Inaccuracy	100.0u		
	16		× TC
Statistics			
Algorithm	Averaging		
Min Samples	200		Sample
Min Time (s)	100.0m		
	25		× TC
Options			
Phase Unwrap	0	Spectral Density	0
Sinc Filter	0	AWG control	0

Normalised output

- Normalised Load (0-1) as normalisation ratio for VOUT
- First resonance – system Servocell response
- Normalised for constant load.
- We can see bender+loadcell resonance at ~ 200Hz
- Preload affects the f_r as expected



Bender excitation mechanical 2



Strain calculations

5mN ac max load

Deflection: $\delta = PL^3/3EI$

L=length cantilever 34mm,

Point of load, a=1mm from end.

P=load in N, 0.6mN at resonance

E = modulus = 80 GPa (PZT)

I = area moment of inertia: = $h.w^3 / 12$

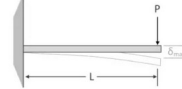
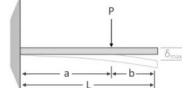
h=1.2mm, w=10mm

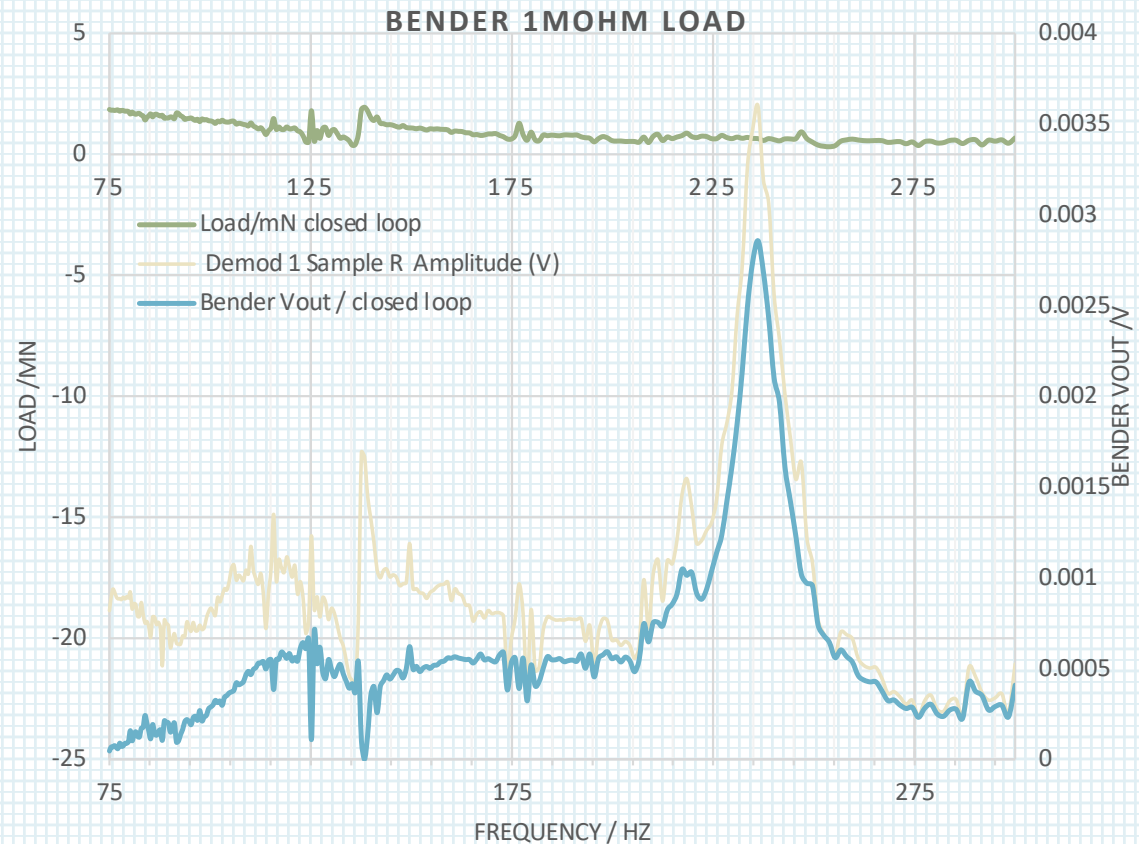
Max deflection: $\delta = 0.08767\mu\text{m}$

Max voltage output: measured: 0.003V

Typical 1000 μm deflection indicates

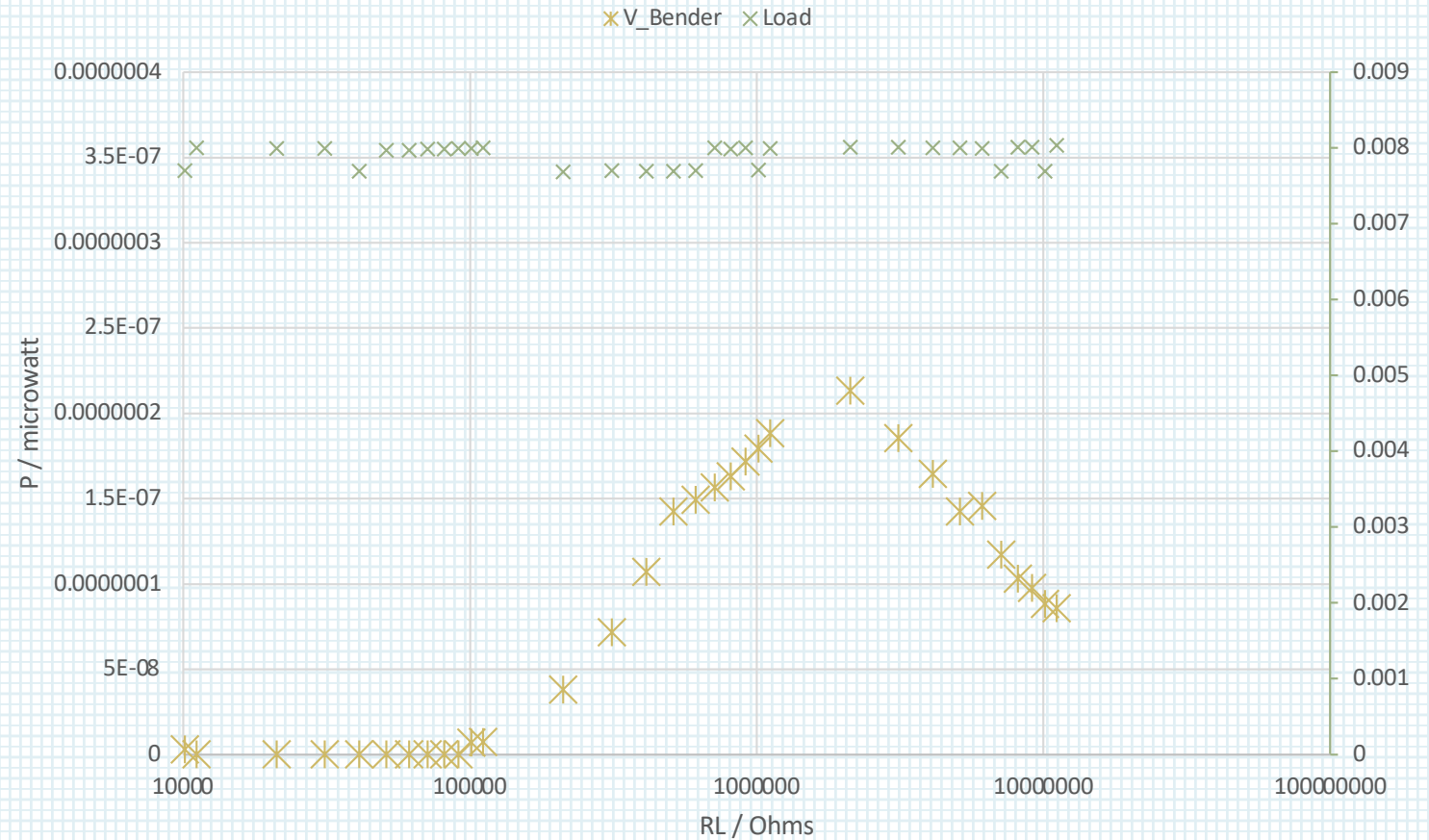
typical voltages 1M Ω shunt \sim 34V

Beam and load cases	Maximum Beam Deflection
	$\delta_{max} = \frac{PL^3}{3EI}$
	$\delta_{max} = \frac{Pa^2(3L - a)}{6EI}$



Matching Loads: Resistance Load

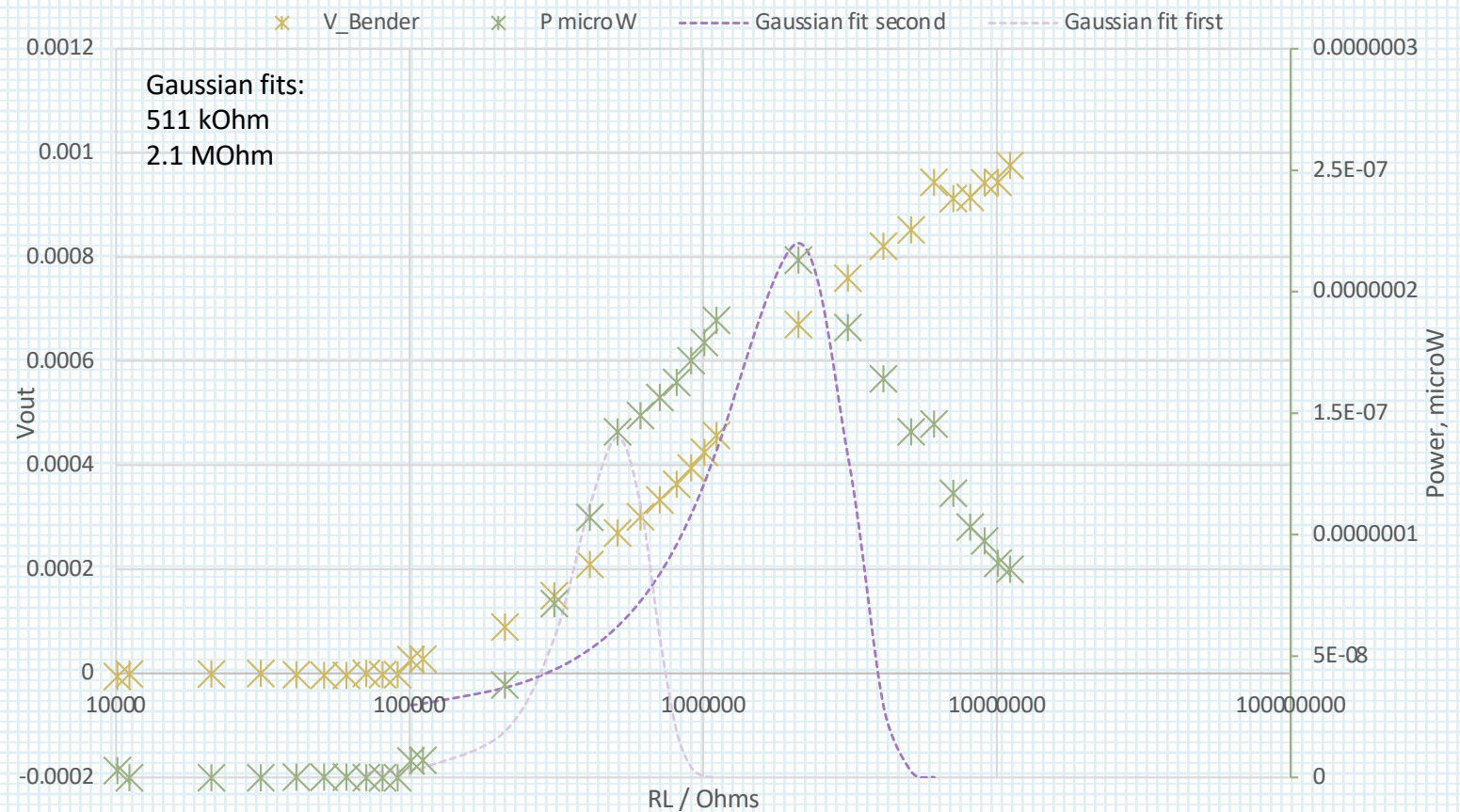
Outcome: Optimum
power transfer into
 $R_L \sim 2.1\text{M}\Omega$



- Set freq at resonance freq? No.
- Stay away from resonance? Flat insensitive to load at about 35Hz. Yes.
- Run freq sweeps for each RL. Explore the sensitivity of fr with frequency.
 - Minimal.
- Load the bender cantilever with R_L .
- Plot, $P=V^2/R_L$ against R_L

Resistive Loads

- Asymmetrical bender actuator: two distinct power load matches
- Points to the possibility for considering harvesting each element separately.
 - Phase is not exactly 180° between each element.
 - Difference amplifier does not then maximise output.



Measurement Summary

Calibrated Energy Harvesting Rig

- Voltage-calibrated Lock-in amplifier
 - Loadcell calibrated with NIST standard
 - Resistance load decade box – calibrated at source
 - DVM calibrated - Keysight calibration
 - Buffer amplifier gain calibration via voltage source and calibrated DVM
- Bender loading conditions & assumptions
 - Composite modulus 80GPa
 - Sample dimensions using calibrated vernier callipers
 - Displacement at resonance calculated ($\sim 228\text{Hz}$)
 - Acceleration: $a = dv/dt = d^2z/dt^2$
 - $a = |z_0| \cdot (2 \cdot \pi \cdot f)^2 \sim 0.18\text{ms}^{-2} \sim 0.02g$
 - Set acceleration: 1g
 - Thus, $|z| \sim 50 * |z_0|$
 - Expect $V \sim 0.2V$
 - Increase acceleration $|z| \sim 1\text{mm}$
 - $a \sim 200g$
 - $V_{\text{out}} \sim 31V$
 - Optimize output with matched R_L
 - $V \sim$ experimental conditions $\sim 7\text{mV}$
 - Gain $\sim 7/3$
 - $V_{\text{out}} \sim 7 \times 31/3 \sim 72V$ pp.
- Errors and uncertainty analysis:
 - Approach: Use of partial differentials for compounding of individual errors through analysis of linear equations.
 - Largest error: calculation of displacement.

Executive summary -Table

Parameter	Measured values	Displacement Calculated values $\pm 10\%$	Acceleration /g $\pm 15\%$	V_{out} ($\pm 0.01mV$) measured	V_{out} $\pm 15\%$ expectation	Assumptions –at cantilever mechanical resonance
δ , displacement	Force, V_{out}	0.088 μm	0.02	3 mV		$E \sim 80GPa$, resonance (228Hz), linear bending moment.
δ , displacement	-	1000 μm	200	-	31 V	Ratiometric output with deflection. $R_L = 1M\Omega$
V_{out}	-	1000 μm	200	-	72 V	$R_L = 2.2M\Omega$ ideal matched. Gain increase=7/3
V_{out}	-	4.4 μm	1	-	0.4 V	$R_L = 2.2M\Omega$ ideal matched. Gain increase=7/3